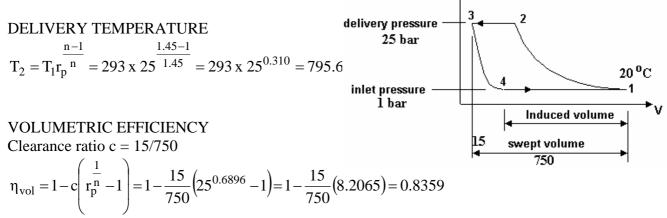
THERMODYNAMICS 201 2004

- 5. A single-stage air compressor has a clearance volume of $15 \times 10^{-6} \text{ m}^3$ and a swept volume of $750 \times 10^{-6} \text{ m}^3$. Air enters the compressor at a temperature of 20° C and a pressure of 1 bar. The delivery pressure is 25 bar and the compressor speed is 600 rev/min. Assume for the compression and expansion strokes that the polytropic indices are identical and equal to 1.45 respectively, and the gas constant for air is 0.287 kJ/kgK.
 - (a) Sketch the ideal indicator diagram.
 - (b) Determine
 - (i) The delivery temperature.
 - (ii) The mass flow rate.
 - (iii) The indicated power.

(c) Show how an actual indicator diagram would differ from the ideal diagram and explain why.

The ideal cycle is as shown.



Induced volume = $0.8359 \times 750 = 626.9 \text{ cm}^3$

Induced flow rate = $626.9 \times 10^{-6} \times 600 \text{ rev/min} = 0.376 \text{ m}^3/\text{min}$ Mass flow rate

 $m = \frac{pV}{RT} = \frac{1x10^5 x \ 0.376}{287 x \ 293} = 0.447 \ \text{kg/min} = 0.007455 \ \text{kg/s}$

INDICATED POWER

There are various ways to find this. A derived formula for the standard cycle is as follows.

$$P = mRT_{l}\left(\frac{n}{n-1}\right)\left\{r_{p}^{\frac{n-1}{n}} - 1\right\} = 0.007455 \text{ x } 287 \text{ x } 293\left(\frac{1.45}{0.45}\right)\left\{25^{0.310} - 1\right\} = 3465 \text{ W}$$

or

$$P = \frac{nmR}{n-1}(T_2 - T_1) = \frac{1.45 \times 0.007455 \times 287}{0.45}(795.6 - 293) = 3465 \text{ W}$$

In practice there is restriction when the air is being sucked in and pushed out and the valves move on their springs so actual cycle is more like this.

