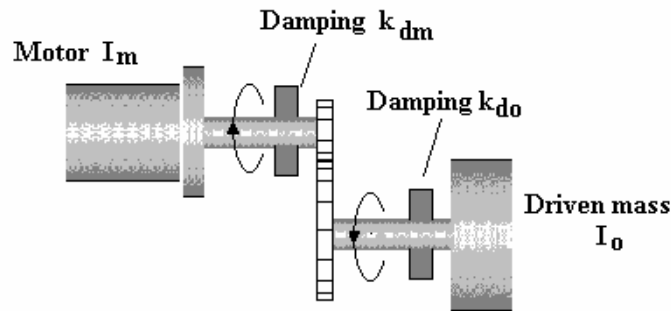


A motor produces a torque T that is related to angle of rotation θ by $T = 500 \sin \theta$ over half a revolution and then repeated. The motor is connected to a load through a 3/1 reduction gear box. The moment of inertia of the motor is 0.2 kg m^2 and the load is 1 kg m^2 .

- (a) Construct a dynamically equivalent system referred to the motor shaft.
- (b) Determine the inertia of the flywheel to be added to the motor shaft in order control the load speed at $250 \text{ rev/min} \pm 1\%$.
- (c) Calculate the maximum angular acceleration of the load during each revolution.

SOLUTION

- (a) When a mass is rotated through a gear system, the affect of the inertia is dramatically altered. Consider a motor coupled to a load through a speed changing device such as a gear box. In this case there is no damping.



θ_m is the motor rotation and θ_o the output rotation. The gear ratio is $G_r = \theta_o/\theta_m$

Since this is a fixed number and is not a function of time, the speed and acceleration are also in the same ratio.

$$G_r = \omega_o/\omega_m \quad \omega \text{ is the angular velocity}$$

$$G_r = \alpha_o/\alpha_m \quad \alpha \text{ is the angular acceleration.}$$

The power transmitted by a shaft is given by $\text{Power} = \omega T$. If there is no power lost, the output and input power must be equal so it follows that

$$\omega_m T_m = \omega_o T_o \text{ hence } T_m = \omega_o T_o / \omega_m = G_r T_o$$

Consider the inertia torque due the inertia on the output shaft I_o .

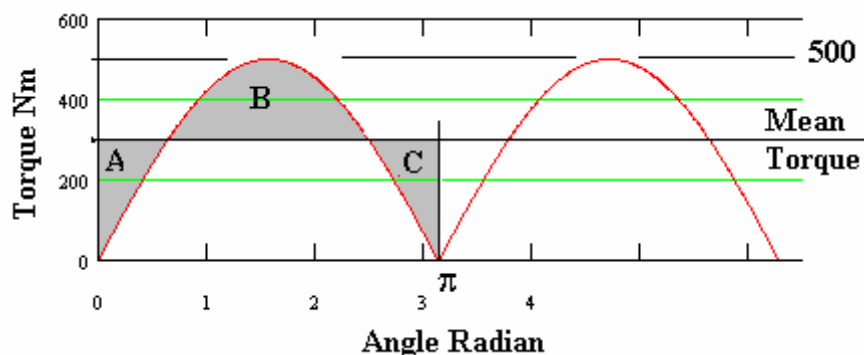
$$T_o = I_o \alpha_o = I_o \alpha_m \times G_r$$

$T_m = T_o \times G_r = I_o \alpha_m \times G_r^2$ This is the output inertia torque referred to the motor. In addition we have the inertia torque of the motor itself $I_m \alpha_m$. This must be added so:

$$T_m = I_m \alpha_m + I_o \alpha_m \times G_r^2$$

$T_m = \alpha_m (I_m + G_r^2 I_o)$ and $(I_m + G_r^2 I_o)$ is the effective moment of inertia I_e referred to the motor shaft.

- (b)



The coefficient of fluctuation of speed $\phi = \Delta\omega/\omega = 2\%$ or 0.02 (for both motor and load)

The coefficient of fluctuation of energy β

$\beta = \text{max fluctuation in energy/work done per cycle}$ $\beta W = \text{max fluctuation in energy}$

$\beta W = I\omega^2\phi$ (see tutorial on fly wheels and TM diagrams)

$\omega = \text{mean angular velocity.}$

Motor speed = 750 rev/min $\omega = 2\pi \times 750/60 = 78.54 \text{ rad/s}$

$W = \text{work done per cycle. In this case we will choose } \frac{1}{2} \text{ revolution as the cycle since it is repeated every } \frac{1}{2} \text{ revolution.}$

$$W = \int_0^{\pi} 500 \sin\theta \, d\theta = 500 [\cos\theta]_0^{\pi} = 1000 \text{ J}$$

$$\text{Mean Torque} = W/\text{base} = 1000/\pi = 318.31 \text{ Nm}$$

The angle where the graph cuts the mean is $\sin^{-1}(318.31/500) = 39.54^\circ$ (0.69 rad) and $180^\circ - 39.54^\circ = 140.46^\circ$ (2.451 rad).

$$\text{The area represented by B is found from: } W = \int_{0.69}^{2.45} 500 \sin\theta \, d\theta = 500 [\cos\theta]_{0.69}^{2.45} = 771.178 \text{ J}$$

It follows that areas A and B are both half this and equal to 385.59 J

The maximum fluctuation of energy is from (mean - 385.59) to (mean + 385.59) so $\beta W = 771.178$

$\beta = I\omega^2\phi/W$ so $I = \beta W/\omega^2\phi = 771.178/(78.54^2 \times 0.02) = 6.251 \text{ kg m}^2$

$$I_T = I_m + Gr^2I_o + I$$

$$6.251 = 0.2 + (1/3)^2 \times 1 + I$$

$$6.251 = 0.311 + I \quad I = 5.94 \text{ kg m}^2$$

(c) $T = I\alpha$ $\alpha = \text{angular acceleration of the motor.}$

The maximum torque is 500 N m so the maximum $\alpha = 500/6.251 = 80 \text{ rad/s}^2$

The maximum acceleration of the load is reduced by the gear ratio of 3 so the maximum acceleration of the load is $80/3 = 26.7 \text{ rad/s}^2$